

Rather than resonance, flapping wing flyers may play on aerodynamics to improve performance

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Saving energy and enhancing performance are secular preoccupations shared by both nature and human beings. In animal locomotion, flapping flyers or swimmers rely on the flexibility of their wings or body to passively increase their efficiency using an appropriate cycle of storing and releasing elastic energy. Despite the convergence of many observations pointing out this feature, the underlying mechanisms explaining how the elastic nature of the wings is related to propulsive efficiency remain unclear. Here we use an experiment with a self-propelled simplified insect model allowing to show how wing compliance governs the performance of flapping flyers. Reducing the description of the flapping wing to a forced oscillator model, we pinpoint different nonlinear effects that can account for the observed behavior—in particular a set of cubic nonlinearities coming from the clamped-free beam equation used to model the wing and a quadratic damping term representing the fluid drag associated to the fast flapping motion. In contrast to what has been repeatedly suggested in the literature, we show that flapping flyers optimize their performance not by especially looking for resonance to achieve larger flapping amplitudes with less effort, but by tuning the temporal evolution of the wing shape (i.e., the phase dynamics in the oscillator model) to optimize the aerodynamics.

Flying animals have long since inspired admiration and fueled the imagination of scientists and engineers. Alongside biologists studying form and function of flapping flyers in nature (1, 2), the last decade has seen an impressive quantity of studies driven by engineering groups using new techniques to develop and study artificial biomimetic flapping flyers (3, 4). The widespread availability of high-speed video and in particular the merging of experimental methods borrowed from fluid mechanics into the toolbox of the experimental biologist have permitted to elucidate various key mechanisms involved in the complex dynamics of flapping flight (see, for example, refs. 5–7).

A recent field of investigation concerns the efficiency of flapping flyers, the major interrogation being about how natural systems optimize energy saving together with performance enhancement. In particular, the passive role of wing flexibility to increase flight efficiency through the bending of flapping wings has attracted a lot of attention. It is commonly agreed that this efficiency enhancement comes from the particular shape of the bent wing, which leads to a more favorable repartition of the aerodynamic forces (see refs. 8 and 9 for an extensive review). For flying animals in air, such as insects, it has been proposed (10–12) that wing inertia should play a major role in competing with the elastic restoring force, compared to the fluid loading. The mechanism governing the propulsive performance of the flapping flyer can therefore be seen at leading order as a two-step process, where the instantaneous shape of the wings is determined by a structural mechanics problem that then sets the moving boundaries for the aerodynamic problem.

From a dynamical point of view, if we consider chordwise bending of a wing with a given flapping signal imposed at the leading edge, the instantaneous shape of the structure is strongly dependent on the phase lag between the forcing and the response

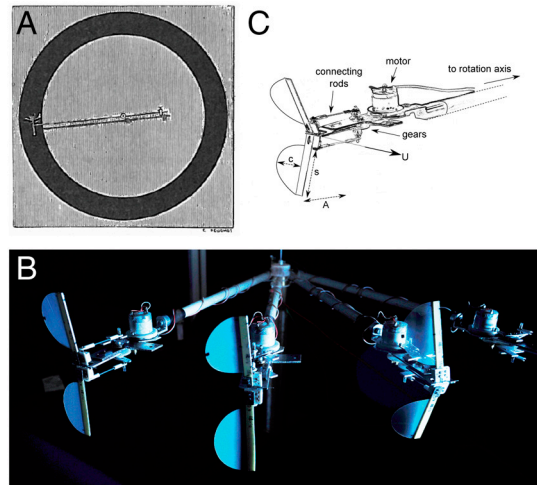


Fig. 1. Experimental setup. (A) Pioneer experiment from Marey (32). (B) Actual setup. (C) Details of the flapping flyer model used for this study.

of the wing (respectively the leading and trailing edges). Recent works (13, 14), by using a simplified model of a flexible wing as a combination of heaving and passive pitching have shown that a transition from enhanced thrust to underperformance occurs for a critical phase value close to the resonant frequency of the system. Those last observations would sustain the commonly invoked argument suggesting that flapping flyers could take advantage of a structural property to save energy by matching the resonant frequency of their compliant wings to the wingbeat frequency (13, 15–17). In nature this phenomenon has been observed in particular for undulatory swimming fish or other swimmers that use deforming propulsive structures, such as jellyfish or scallops (see ref. 18 and references therein). In the case of insects, however, the few available observations (especially for large species) report wingbeat frequencies far below the natural resonant frequencies (19–22). Recent experiments using a self-propelled model with large-flapping-amplitude elastic wings (12) are consistent with the latter, because the propulsive efficiency of the model peaks for a flapping frequency lower than the primary linear resonance of the wings. Fully predicting the wing beat rate as the undamped resonant frequency of a linear oscillator (see, for example, ref. 15) should be therefore taken with reserve. Superharmonic nonlinear resonances have also been invoked (23), suggesting that flying animals may effectively flap their wings far below the primary resonance while increasing their performance. This is probably one mechanism among others

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In this paper we address these questions using the experimental self-propelled flapping-wing model with elastic wings described in ref. 12. Exploring a wide range of bending rigidities, we show that, in the simplified context of chordwise-compliant wings, the performance optima of the system are far from being set by a simple resonant condition. We develop a nonlinear one-dimensional beam model for the bending wing that is reduced to a forced oscillator model suitable to study different nonlinear effects. In particular, a set of cubic nonlinearities coming from the clamped-free beam equation and a quadratic damping term representing the fluid drag associated to the fast flapping motion permit to account for the observed behavior. We show that the nonlinear nature of the fluid damping is an essential feature to determine the phase lag that leads to an increase/decrease of the efficiency. As a whole fluid–solid interaction process leading to propulsion, we provide evidence that flapping flyers may optimize their performance not by especially looking for resonance but by using passive deformation to streamline the instantaneous shape of the wing with the surrounding flow.

Physical Quantities. This specific self-propelled system of flyer used here (see Fig. 1) allows to measure various averaged quantities for different wing rigidities (see ref. 12 for details): the cruising speed U as the device is allowed to turn around, and the thrust force F_T whose direct measurement is performed at a fixed station (see Fig. 2A and B). The product of both quantities gives the averaged aerodynamic thrust power $P_T = UF_T$. In both cases, the power consumption P_i is measured. On the other hand, we performed a precise dynamical study of the flapping wing. For each set of parameters (A_ω, f, B), the phase and amplitude of the trailing edge, with respect to the forcing flapping motion (defined as $u_\omega = A_\omega \omega f \cos(\omega f t)$), has been measured using a fast cadenced camera [1,000 frames per second (fps)] in both air and vacuum. It is important to recall that for this setup, and more generally for flapping flyers in air, the main bending motor of the flexible wings is wing inertia (10–12). The competition between the wing inertia and the elastic restoring force is captured by the scaled elastoinertial number \mathcal{N}_{ei} (12):

The first expression is a direct comparison between both the moments of inertial and elastic forces. Interestingly, this number can also be expressed as a function of the ratio between the forcing and relaxation frequencies times the nondimensional forcing amplitude of the driving motion, which allows to express directly the bending rate as function of a nondimensional oscillator forcing term. The second expression is therefore useful to explore the nearness of the resonance and will be used to analyze the experimental data in this paper. Results will be therefore displayed as a function of the reduced frequency $\tilde{\omega}_f = (\omega_f/\omega_0) = \tilde{A}_\omega^{-1/2} \mathcal{N}_{\text{ci}}^{1/2}$, where $\tilde{A}_\omega = \frac{A_\omega}{L}$ is the reduced flapping amplitude. To compare the aerodynamic performance in all the experiments, both the thrust force and cruising speed were rendered nondimensional using the appropriate scalings $f_T = F_T L/B$ and $u = U/A_\omega \omega_f$ (see Fig. 2 C and D), where $u_\omega^{\text{max}} = A_\omega \omega_f$ is the maximum flapping velocity. The nondimensional powers (displayed in Fig. 2 E and F) then read $p_T = U F_T L / B A_\omega \omega_f$ and $p_i = P_i L / B \omega_f$.

Figure 3 consists of six panels (A-F) showing the dependence of various quantities on frequency f and normalized frequency $\bar{\omega}_f$.

- Panel A:** Force F_T (N) vs frequency f (Hz). The y-axis ranges from 0 to 0.04. The x-axis ranges from 10 to 30. Multiple colored lines show different force profiles.
- Panel B:** Velocity U (m.s⁻¹) vs frequency f (Hz). The y-axis ranges from 0 to 3. The x-axis ranges from 10 to 30. An arrow points to a region labeled "near rigid case".
- Panel C:** Force f_T vs normalized frequency $\bar{\omega}_f$. The y-axis ranges from 0 to 4. The x-axis ranges from 0 to 1. A vertical gray shaded region is centered around $\bar{\omega}_f \approx 0.8$.
- Panel D:** Velocity u vs normalized frequency $\bar{\omega}_f$. The y-axis ranges from 0 to 0.8. The x-axis ranges from 0 to 1. A vertical gray shaded region is centered around $\bar{\omega}_f \approx 0.8$.
- Panel E:** Probability density p_t vs normalized frequency $\bar{\omega}_f$. The y-axis ranges from 0 to 2. The x-axis ranges from 0 to 1. A vertical gray shaded region is centered around $\bar{\omega}_f \approx 0.8$.
- Panel F:** Probability density p_i vs normalized frequency $\bar{\omega}_f$. The y-axis ranges from 0 to 800. The x-axis ranges from 0 to 1. A vertical gray shaded region is centered around $\bar{\omega}_f \approx 0.8$.

Fig. 2. (A) Thrust force [nondimensional (C)] and (B) cruising speed [nondimensional (D)] as a function of the forcing frequency [reduced forcing frequency $\bar{\omega}_f$ for (C) and (D)]. Nondimensional (C) thrust (p_T) and (D) input (p_i) powers as a function of $\bar{\omega}_f$. The gray area represents the nearness of the optimum region, the dashed line indicates the location of the reduced natural frequency of the wing (linear resonance).

Up to a certain flapping frequency, the more flexible wings outperform the rigid linear $U(f)$ relationship (see also ref. 24). The measurements for the two most flexible wings then evidence the appearance of an underperformance regime in which both F_T and U lie below the rigid wing case. Looking now at the nondimensional thrust power, the data from all wings collapse on a single curve with a clear performance peak, which agrees with what has been observed in refs. 13 and 14 for heaving/pitching systems. An important point is that the maximum in performance does not take place at the resonant frequency, but much below (around $0.7\omega_0$). Moreover, the nondimensional thrust power at $\hat{\omega}_f = 1$ (see dashed line in Fig. 2E) is even more than 4 times lower than the optimum value. At last, we remark that there is also no sign of a resonant behavior in the consumed power curve (Fig. 2F). It is worth noting that because F_T and U are respectively measured in a fixed and moving configuration, the flow physics in the nearness of the wing may be different (see for instance discussion in ref. 25). This difference shows in the small discrepancy between the transitions to underperformance of the velocity and the thrust force, which should be correlated if both measurements were performed in the moving configuration. It is however possible to determine a region where the optimum lies, rather than an exact location. The width of this region is estimated from the decoherence of both force and cruising velocity optima, as can be seen in Fig. 2 C and D. We have thus introduced a background gray shaded area standing for the “error” due to the incoherence of measuring quantities at different configurations. This gray shaded area is displayed for all the results all along this work.

Thus, the equation governing the motion of the nonlinear flexural oscillations of clamped-free beam writes (27):

$$EIW'''' + \mu\ddot{W} = -EI(W'W'^2 + W''W'^2)' - \frac{\mu}{2} \left[W' \int_L^x \frac{\partial^2}{\partial t^2} \left[\int_0^x W'^2 dx \right] dx \right]', \quad [2]$$

where the prime and dot notations refer to the derivatives with respect to space and time respectively, W is the transversal local displacement, E the Young modulus, I the second moment of inertia and μ the mass per unit of length. Writing W as $W(x,t) = w(x,t) + w_0(t)$, where $w_0(t)$ is the driving motion defined by $w_0(t) = A_\omega \cos(\omega_f t)$, and using the nondimensional quantities for space and time $\tilde{w} = \frac{w}{L}$; $\tilde{x} = \frac{x}{L}$; $\tilde{t} = \frac{t}{\tau}$ with $\tau = (\frac{\mu}{EI})^{1/2} L^2$, Eq. 2 reads:

$$\tilde{w}'''' + \ddot{\tilde{w}} = -(\tilde{w}'\tilde{w}'^2 + \tilde{w}''\tilde{w}'^2)' - \frac{1}{2} \left[\tilde{w}' \int_1^{\tilde{x}} \frac{\partial^2}{\partial \tilde{t}^2} \left[\int_0^{\tilde{x}} \tilde{w}'^2 d\tilde{x} \right] d\tilde{x} \right]' - \tilde{A}_\omega \ddot{\tilde{w}}_0, \quad [3]$$

which has to satisfy the clamped-free boundary conditions $\tilde{w}(0,\tilde{t}) = \tilde{w}'(0,\tilde{t}) = \tilde{w}''(1,\tilde{t}) = \tilde{w}'''(1,\tilde{t}) = 0$. The last term on the right hand side in Eq. 3, $-\tilde{A}_\omega \ddot{\tilde{w}}_0 = \tilde{A}_\omega \tilde{\omega}_f^2 \cos(\tilde{\omega}_f \tilde{t}) = \mathcal{N}_{ei} \cos(\tilde{\omega}_f \tilde{t})$, is a forcing term due to the wing inertia whose amplitude is given by the elastoinertial number and that is dependent on the square of the driving frequency.

The next step is to set apart the spatial dependence by projection of Eq. 3 onto the complete set of eigenfunctions defined by the linear part. The displacement is expanded as $w(x,t) = \sum_1^\infty X_p(t) \Phi_p(x)$ (see ref. 28) where Φ_p are the nondimensional linear modes for clamped-free beams that are not recalled here for the sake of brevity. The problem then writes (the $\tilde{\cdot}$ have been removed for simplicity):

$$\ddot{X}_p + X_p = - \sum_{i,j,k=1}^N h_{ijk}^p X_i X_j X_k - \sum_{i,j,k=1}^N f_{ijk}^p (X_i X_j \ddot{X}_k + X_i \dot{X}_j \dot{X}_k) + F_p(t), \quad [4]$$

where h_{ijk}^p and f_{ijk}^p are determined by:

$$h_{ijk}^p = \int_0^1 (\Phi_i' \Phi_j' \Phi_k' + \Phi_i''' \Phi_j' \Phi_k') \Phi_p dx \quad [5]$$

$$f_{ijk}^p = \int_0^1 \left[\Phi_i' \int_1^x \int_0^u \Phi_j'(y) \Phi_k'(y) dy du \right]' \Phi_p dx. \quad [6]$$

The projection of the forcing term on the p th mode, F_p , writes at the trailing edge:

$$F_p = \tilde{A}_\omega \tilde{\omega}_f^2 \Phi_p(1) \int_0^1 \Phi_p(x) dx. \quad [7]$$

As the propulsive regimes observed in this work lie below the first relaxation frequency of the wing, we assume that the response of the wing is mainly governed by the first eigenmode. Hence, Eq. 4 can be considerably simplified and reduces for the only mode 1 to:

$$\ddot{X} + X = -h_{111}^1 X^3 - f_{111}^1 (X^2 \ddot{X} + X \dot{X}^2) + F_1(t). \quad [8]$$

A crucial feature is now to choose a damping term to this dynamical system. During a stroke cycle, the wing follows very fast motions involving high local Reynolds numbers ($Re_\omega = \frac{A_\omega \omega_f L}{\nu_{air}} \in [1,000, 10,000]$), which prompt us to include a nonlinear quadratic fluid drag term (29) in addition to the classical linear viscous friction law. The damping is then chosen as a combination of linear and nonlinear terms as follows:

$$\Xi(X, \dot{X}) = \xi \dot{X} + \xi_{nl} |\dot{X}| \dot{X}. \quad [9]$$

The linear and nonlinear coefficients ξ and ξ_{nl} are estimated studying the impulse response for each wing (26) (see *Materials and Methods* for details). The solution of Eq. 8 including damping is

determined by using a classical multiple scale method at first order (see ref. 26). To this end, we introduce a small parameter ϵ and a detuning parameter $\sigma = (\tilde{\omega}_f - 1)/\epsilon$. The problem to be solved reads:

$$\ddot{X} + X = -\epsilon(h_{111}^1 X^3 + f_{111}^1 (X^2 \ddot{X} + X \dot{X}^2) + \Xi(X, \dot{X}) + F_1(t)). \quad [10]$$

According to the multiple scales theory, we express the solution in terms of different time scales as $X = X_0(t_0, t_1) + \epsilon X_1(t_0, t_1) + \dots$, where $t_0 = t$ and $t_1 = \epsilon t$ are respectively short (relative to the oscillation of the wing) and long times scales. The system at order ϵ^0 is $\partial_{t_0}^2 X_0 + X_0 = 0$ and gives the straightforward solution $X_0 = A(t_1) e^{it_0} + A^*(t_1) e^{-it_0}$ where A and A^* are complex functions.

At order ϵ^1 , we obtain:

$$\partial_{t_0}^2 X_1 + X_1 = -h_{111}^1 X_0^3 - f_{111}^1 (X_0^2 \ddot{X}_0 + X_0 \dot{X}_0^2) - \Xi(X_0, \dot{X}_0) - 2\partial_{t_1} X_0 + F_1 \cos(t_0 + \sigma t_1). \quad [11]$$

Using the expression of X_0 found at order ϵ^0 into Eq. 11, an equation for A is obtained by elimination of the secular terms:

$$A^2 A^* (3h_{111}^1 - 2f_{111}^1) + i \left(2\partial_{t_1} A + \xi A + \frac{4\xi_{nl}}{3\pi} |A| A \right) = \frac{1}{2} F_1 e^{i\sigma t_1}, \quad [12]$$

where the prefactor $\frac{4}{3\pi}$ in front of the nonlinear damping coefficient is obtained during the special integration over one period of the Fourier expansion of the function $\dot{X}_0 |\dot{X}_0|$ (see ref. 26). As can be seen, Eq. 12 is a characteristic equation of a forced damped oscillator with cubic nonlinearities. At last, substituting the polar form $A = \frac{1}{2} a e^{i(\sigma t_1 - \gamma)}$, separating into real and imaginary parts and looking only to the steady-state solutions, we find two relations for the amplitude a and phase γ .

$$(\Gamma_1 a^3 - a\sigma)^2 + \left(\xi a + \frac{4}{3\pi} \xi_{nl} a^2 \right)^2 = \frac{F_1^2}{4} \quad [13]$$

$$\gamma = \arctan \left(\frac{(\xi a + \frac{4}{3\pi} \xi_{nl} a^2)}{\Gamma_1 a^3 - a\sigma} \right), \quad [14]$$

where $\Gamma_1 = \frac{1}{8} (3h_{111}^1 - 2f_{111}^1)$ is the nonlinear cubic term coefficient, which is computed from Eqs. 5 and 6.

Eqs. 13 and 14 closely resemble a classic nonlinear Duffing oscillator except that the forcing amplitude is frequency dependent and that a nonlinear damping term is present.

Discussion

Resonance and Phase Evolution. Predictions of the above model for the parameters of the experiments are plotted in Fig. 4 for both cases in air and vacuum. In addition, for a clear understanding of the underlying dynamics described by Eqs. 13 and 14, a comparison between predictions from a linear model, a nonlinear with linear damping and a nonlinear with nonlinear damping is displayed in Fig. 5 for two flapping amplitudes \tilde{A}_ω . It can be seen that the model based on a single mode is capable of reproducing all the observations made from the experiments both in normal and low density environments. The good agreement between experiments and model allows us to pinpoint some mechanisms underlying the complex mechanisms of flapping flight.

The first concerns the question of resonance: From Fig. 5, it can be observed that the only case (apart from the linear case) exhibiting a slight resonance peak corresponds to relatively small flapping amplitude and damping coefficient (i.e., only linear damping term; see Fig. 5A). Cases for higher amplitude and/or presence of nonlinear damping behave as a nonresonant-like system in the range of flapping frequencies studied. In nonlinear oscillators, it is known that the main effect of the nonlinear term is to distort the resonance curve and shift the resonance peak to higher frequencies (for a hardening coefficient $\Gamma_1 > 0$, as in the present study) (26). An important feature of such nonlinear

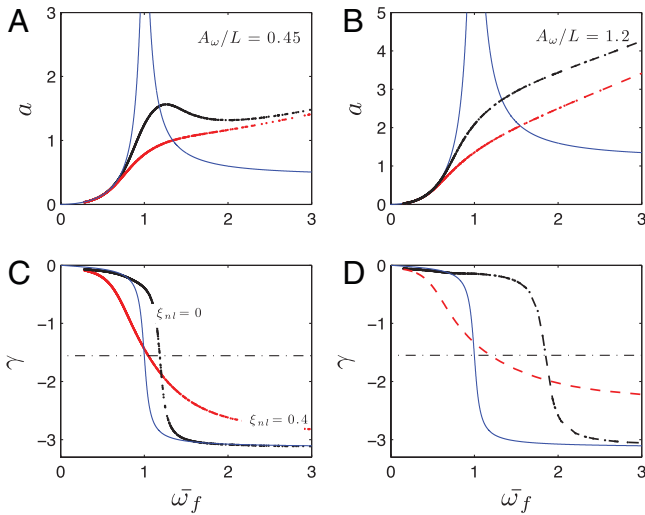


Fig. 5. Dependence of the amplitude a and phase γ with the reduced forcing frequency $\bar{\omega}_f$ for the first mode of a clamped-free beam forced by inertia for two different (high and medium) amplitudes A_ω (chosen arbitrarily for clarity). The blue line corresponds to the linear prediction, the black line to the nonlinear model from Eq 12 with linear damping ($\xi_{nl} = 0$), the red line to the nonlinear model with nonlinear damping ($\xi_{nl} \neq 0$). As can be seen only cases with relatively small flapping amplitude and linear damping can exhibit a slight resonance peak. Greater amplitudes and/or presence of nonlinear damping behave as a nonresonant system in the domain of flyers capabilities. Concerning the phase, models including only linear damping do not produce “useful” phase lag except in the nearness of the phase jump. In contrast, the presence of a nonlinear damping produces a fast and helpful evolution.

systems is that the distortion of the shape of the resonance curve is directly dependent on the amplitude of the excitation. In the present case where the forcing is inertial, the response depends on the square of the forcing frequency (or on the elastoinertial number \mathcal{N}_{ei}), which provides an increase of the amplitude plotted in Fig. 4 independent of an intrinsic resonance mechanism. Hence, we can expect the actual resonance curve of the system to be all the more distorted that the flapping frequency increases. Another feature that makes it difficult for the flapping flyer to benefit from a resonance mechanism is the presence of a geometric saturation due to the finite length of the wing. Always due to the inertia effects, this geometrical saturation will be reached all the more soon that the demand for larger amplitude (i.e., better performance) is increased. Additionally, the presence of strong damping accentuates this behavior by smoothing the value of a possible resonance peak.

The second point is the crucial role of fluid damping in triggering the phase lag that is useful for thrust enhancement. For the phase, shifting the resonance peak as a result of the nonlinear spring in the oscillator model means shifting the phase jump at $\gamma = \pi/2$ to higher frequencies as well. Thus, without air drag, as can be seen in Fig. 5 C and D, the nonlinear evolution of the phase $\gamma(\bar{\omega}_f)$ would be even slower than in the linear case for which the phase evolution is already not especially favorable except in the nearness of the resonance. This is exactly what is observed for the vacuum measurements where the nonlinear damping due to fluid drag is negligible. On the contrary, the presence of a quadratic fluid damping determines a fast increase of the phase lag (and a so a thrust improvement) even from the very first flapping frequencies. This implies of course that strong flapping velocities are a necessary condition for the bending to become efficient (i.e., elasticity will play a minor role if the flapping beat amplitude is not strong enough).

Summarizing, the instantaneous wing shape is given by the two following ingredients: Inertia provokes the bending (gives the amplitude) and damping, by controlling the phase lag, allows this bending to be usefully exploited. Large phase lags will provide

largest bending of the wing at maximum flapping speed, leading to a more favorable repartition of aerodynamic forces.

Optimum. Because classic resonance mechanisms cannot answer it, the question of the performance optimum (or the transition to underperformance) remains unclear. We therefore proceeded to study the kinematics of the wing in the laboratory frame. In particular, we have compared both characteristic angles relative to the global wing motion. The first characteristic angle is dependent on the ratio between the maximal vertical flapping velocity u_{ω}^{\max} and the cruising velocity U and reads: $\phi = \arctan(\omega_f A_\omega / U)$. The angle ϕ is considered as the instantaneous angle of attack of the wing at the midpoint of the heaving cycle, and as can be seen, is directly related to the Strouhal number $St = \omega_f A_\omega / U$ that also determines the performance of flapping flyers (30). We define a second characteristic angle θ as the deflection angle at the trailing edge, also taken at the maximum flapping velocity. This angle is directly related to the phase lag γ , and thus determines to what extent the bending of the wing will be useful in terms of performance. Fig. 6 shows the evolution of the ratio θ/ϕ .

The interesting point is that the location of the performance/under performance transition takes place at $\theta/\phi \approx 1$ (i.e., when both angles point instantaneously at the same direction). Thus, the optimum value of θ does not correspond to the maximum bending experienced by the wing (which should have been a priori the optimal solution) but to the moment when the deflection angle matches the angle of attack as sketched in Fig. 6. For a rigid wing, because θ is fixed ($=0$), the optimization problem is here nonexistent and thrust only depends on the driving frequency (for a given amplitude). With flexibility and according to what has been previously observed, θ starts increasing and tends to align the wing trailing edge with the flow. As discussed earlier, this leads to a more favorable repartition of the aerodynamics forces as sketched in Fig. 6.

However, this argument is only valid if the surrounding flow is totally attached to the wing (i.e., separation occurs only at the trailing edge). A situation where $\theta > \phi$ is strongly subjected to flow separation before the wing trailing edge. In this case, the effective surface relative to the aerodynamic load can be expected to be drastically reduced, leading to a loss of aerodynamic performance. It has to be noticed that the value of $\pi/2$ should be, theoretically, more optimal (i.e., should give more optimal

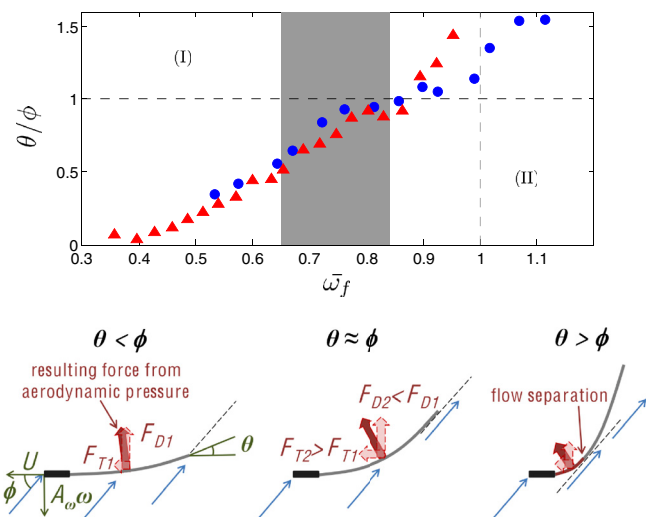


Fig. 6. Evolution of the two characteristic angles of the wing motion θ and ϕ as a function of the reduced driving frequency $\bar{\omega}_f$. Two regimes can be distinguished: (I) The case $\phi < \theta$ corresponding to the performance increasing stage due to a useful phase lag. (II) The $\phi > \theta$ corresponding to the transition to underperformances due to a loss of the effective wing area. The optimum occurs therefore when ϕ and θ point at the same direction (best phase lag).

Table 1. Wing properties

Wing thickness, h (mm)	0.050	0.078	0.130	0.175	0.250	0.360
Mass per unit area μ_s (kg.m ⁻²)	4.50 10^{-2}	10.63 10^{-2}	17.67 10^{-2}	24.12 10^{-2}	34.92 10^{-2}	47.95 10^{-2}
Rigidity B (N.m)	3.34. 10^{-5}	1.83. 10^{-4}	1.02. 10^{-3}	2.26. 10^{-3}	7.31. 10^{-3}	14.00. 10^{-3}
Relaxation frequency f_0 (Hz)	25.4	34.2	62.2	89.5	117.1	160.8
Color label in figures	blue	red	green	yellow	purple	black

bending shapes for useful projection of forces). However, if a separation occurs, the corresponding loss of thrust force (and so cruising speed) will accelerate the decoherence of both angles and hence, will provoke the subsidence of the performance, as has been observed on Fig. 2. The most economic strategy to fly is therefore to set $\theta \approx \phi$ that corresponds to the optimum way to transfer useful momentum.

Conclusions

In this work, we aimed at describing the dynamics governing the performance of flapping flyers. Considering large flapping amplitude and relatively large wings (as for big insect species), we have shown that nonlinear and inertia effects, together with geometric limitation, question the prevailing idea that energy-saving strategies in flapping flight must be related to resonance mechanisms. In search of improving performance, animals may actually stay below the resonance point. Besides, the nonlinear nature of air drag (which implies sufficiently strong flapping amplitudes) seems to be a fundamental ingredient to create the phase lag between the leading and trailing edges of the flapping wing that allows the elasticity energy to be used at its best. One last comment is that the presence of structure resonances for flyers in nature is not invalidated by the mechanism described here. For instance, small insects may not use much elasticity and bending because either their wings are too small or the local Reynolds number is not sufficiently high to produce enough damping, and thus a useful phase lag. However, studies containing a large bank of comparative resonant frequencies and wingbeats of insects or birds being rare in the literature, it is consequently hard to draw any conclusion about the existence of two distinct strategies at this state. According to biologists, resonant mechanisms lie at the muscle level more than in the wing structure itself (see refs. 2 and 31, and references therein) that would strengthen the fact that there is no reason, a priori, for flapping

flyers to look for structural resonance of the wing. Further analysis on such a way would certainly help to discern if there are, or not, universal characteristics for flapping flyers.

Materials and Methods

Experiments. The experimental setup is the same described by Thiria and Godoy-Diana (12), inspired from the pioneer 19th century experiment by Marey (see, for example, ref. 32): A flapping-wing device is attached to a mast that is ball bearing mounted to a central shaft in such a way that the thrust force produced by the wings makes the flyer turn around this shaft (see Fig. 1). A particular attention has been paid to reduce friction losses in the whole system. Wings are made of Mylar semicircles of diameter $S = 2L = 6$ cm. The experimental parameters are the forcing frequency (f), the flapping amplitude (A_w), and the chordwise rigidity of the wings (B) governed by their thickness h . In contrast with the first study reported with this setup (12), the set of wings used here covers a larger range of bending rigidities, from near-rigid to very soft materials. Six pairs of wings have been tested. Their structural properties (thickness, mass, and rigidity) are summarized in Table 1.

Damping Coefficients Estimation. Damping coefficients have been measured by fitting the impulse response of each wing in air. The discrimination between nonlinear and linear coefficients has been achieved by using two different fitting analytical functions on respectively the high (large displacements) and low part (small displacements) of the impulse response curve (see ref. 26 for more details). It has to be noted that the linear damping term $a\xi$ corresponds to structural damping (and viscous fluid damping relative to very small displacements) and is therefore mainly dependent on the only displacement X (i.e., in the wing frame). In contrast, $\frac{4}{3\pi}\xi_{nl}a^2$ is strongly dependent on the global motion of the wing and has therefore to be estimated in the laboratory frame. Thus, at first order, a reasonable corrected approximation for this term is $\frac{4}{3\pi}\xi_{nl}(a + A_w)^2$.

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