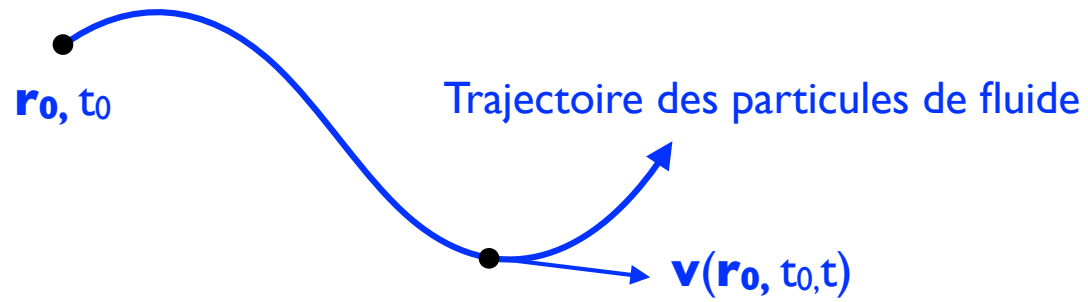


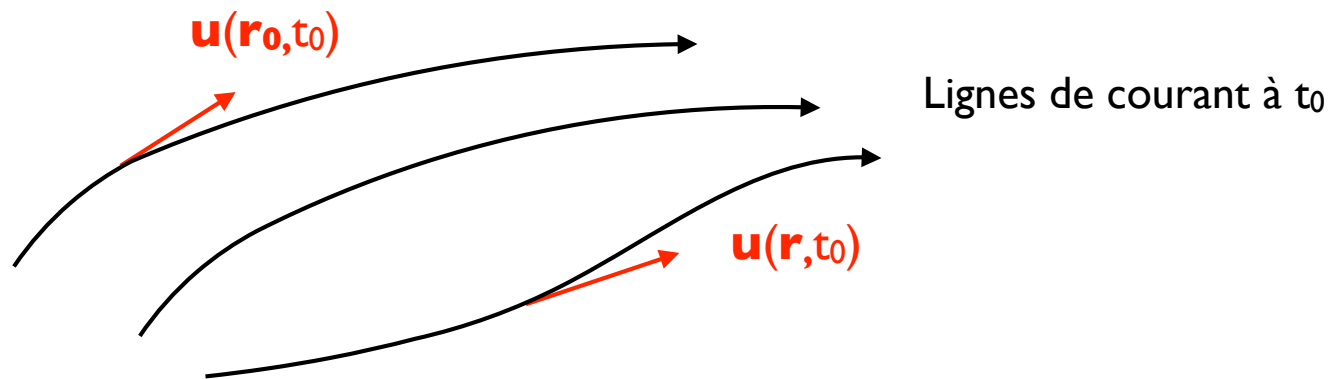
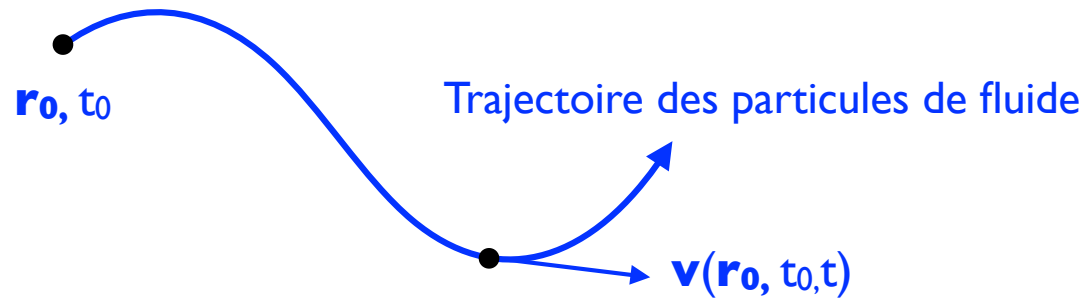
Description des écoulements



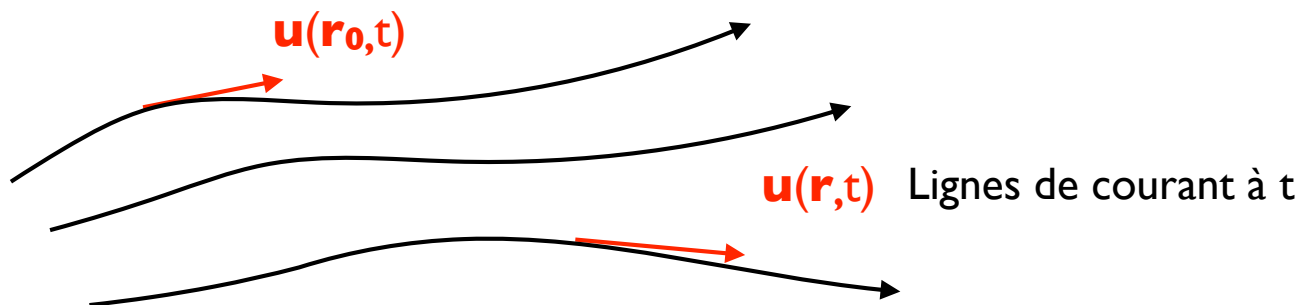
Description lagrangienne

Description des écoulements

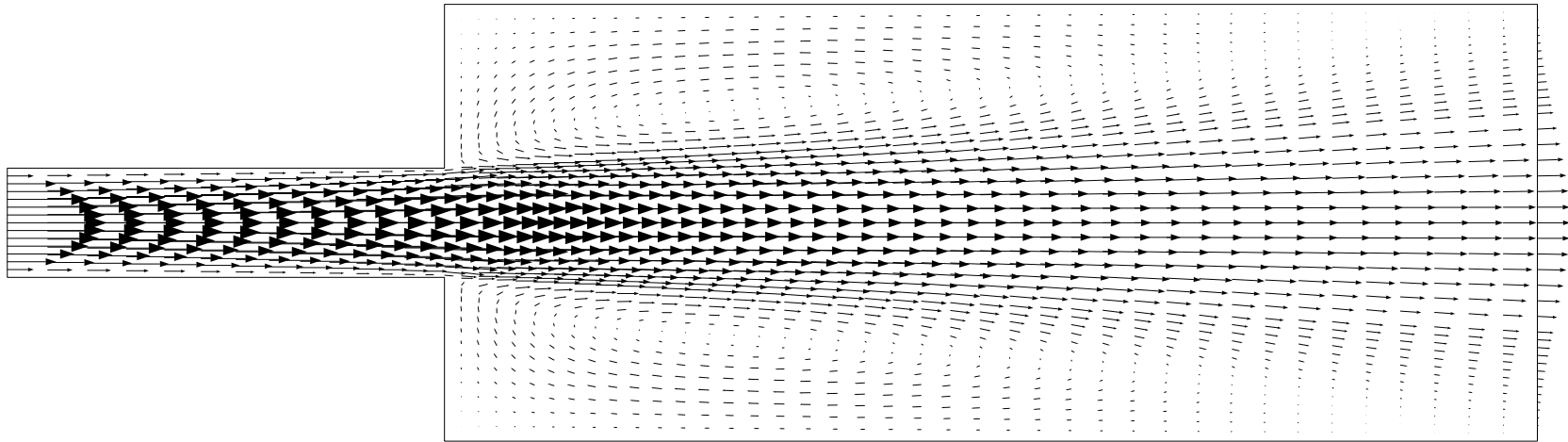
Description lagrangienne



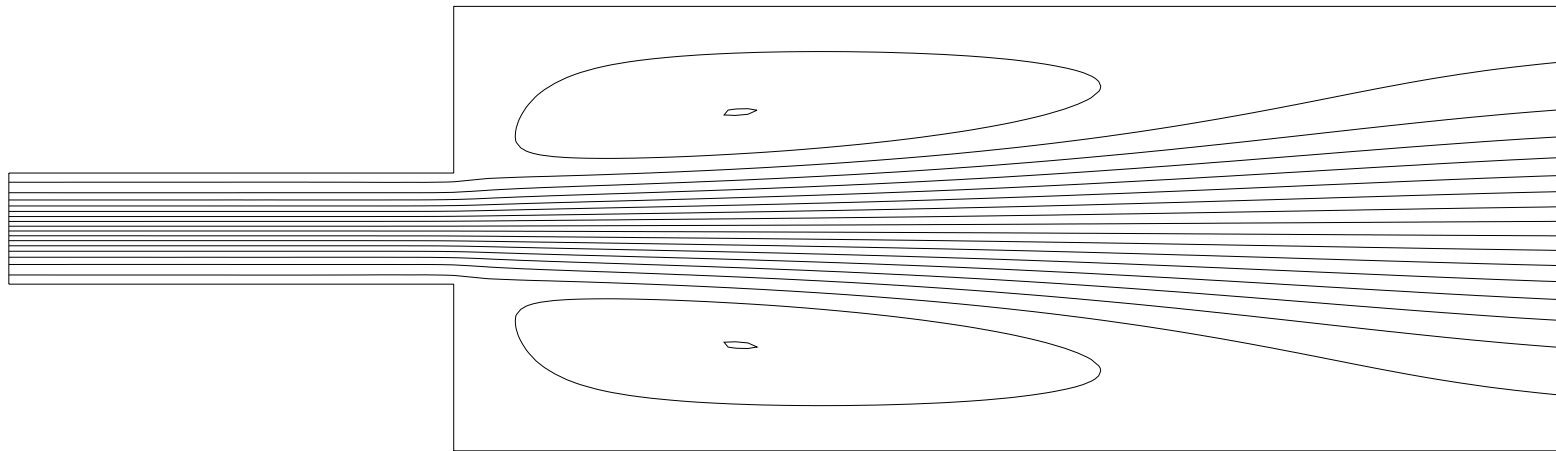
Description eulérienne



Un exemple de champ de vitesse eulérienne

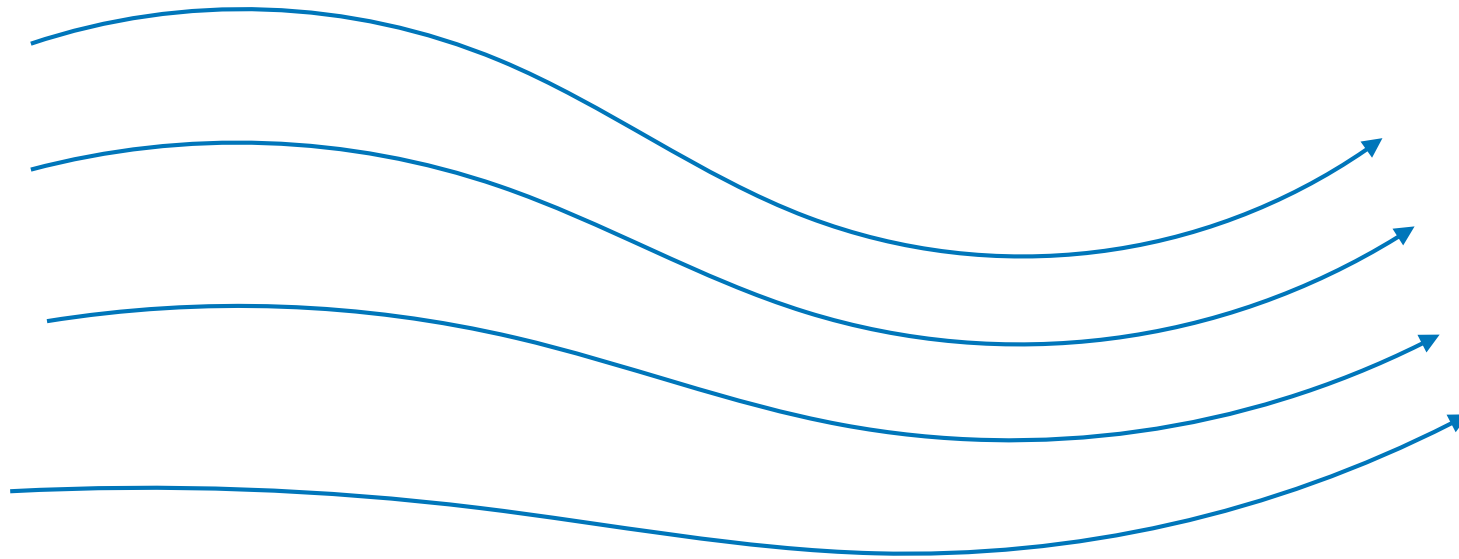


Vecteurs vitesse

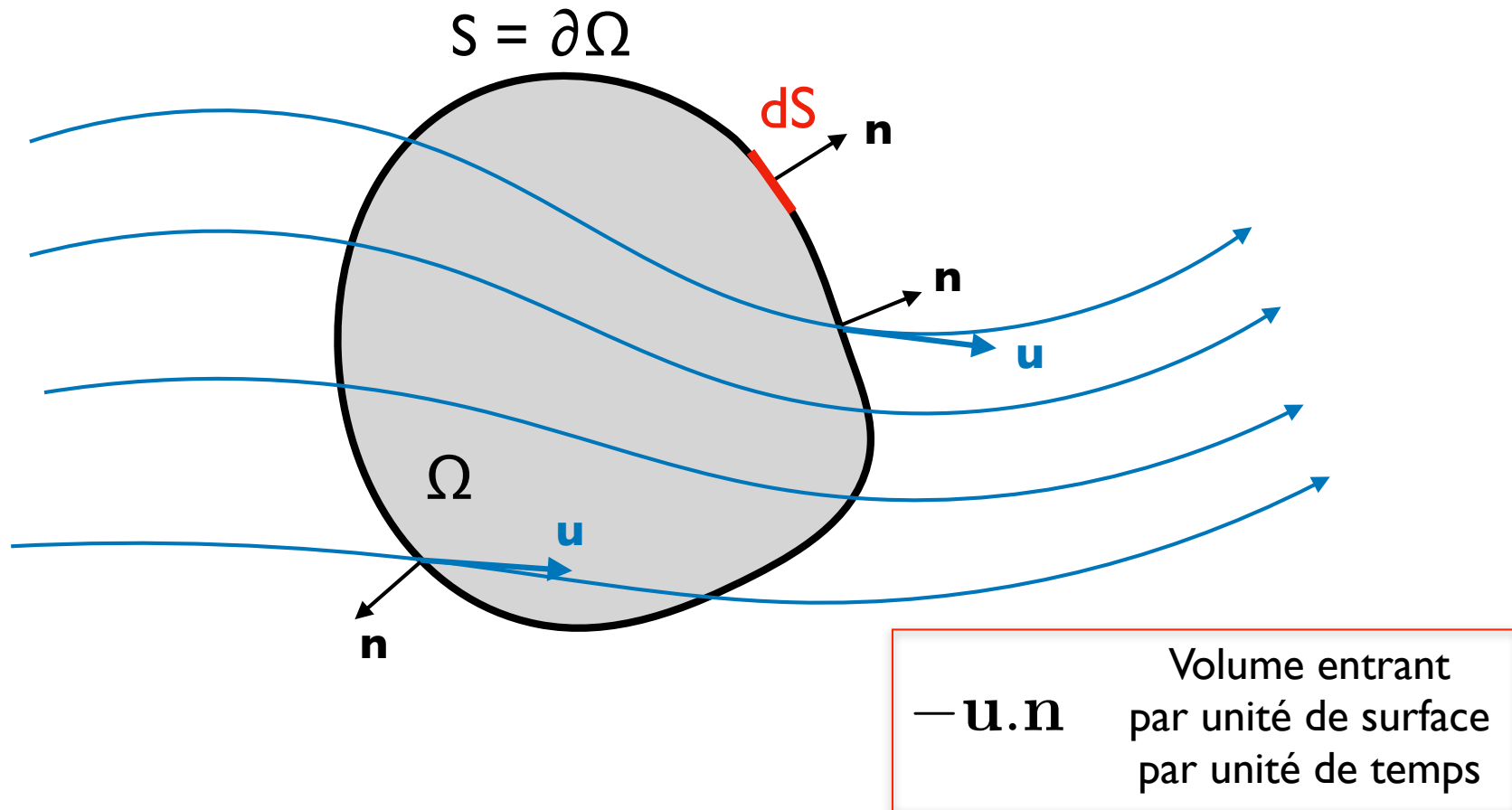


Lignes de courant

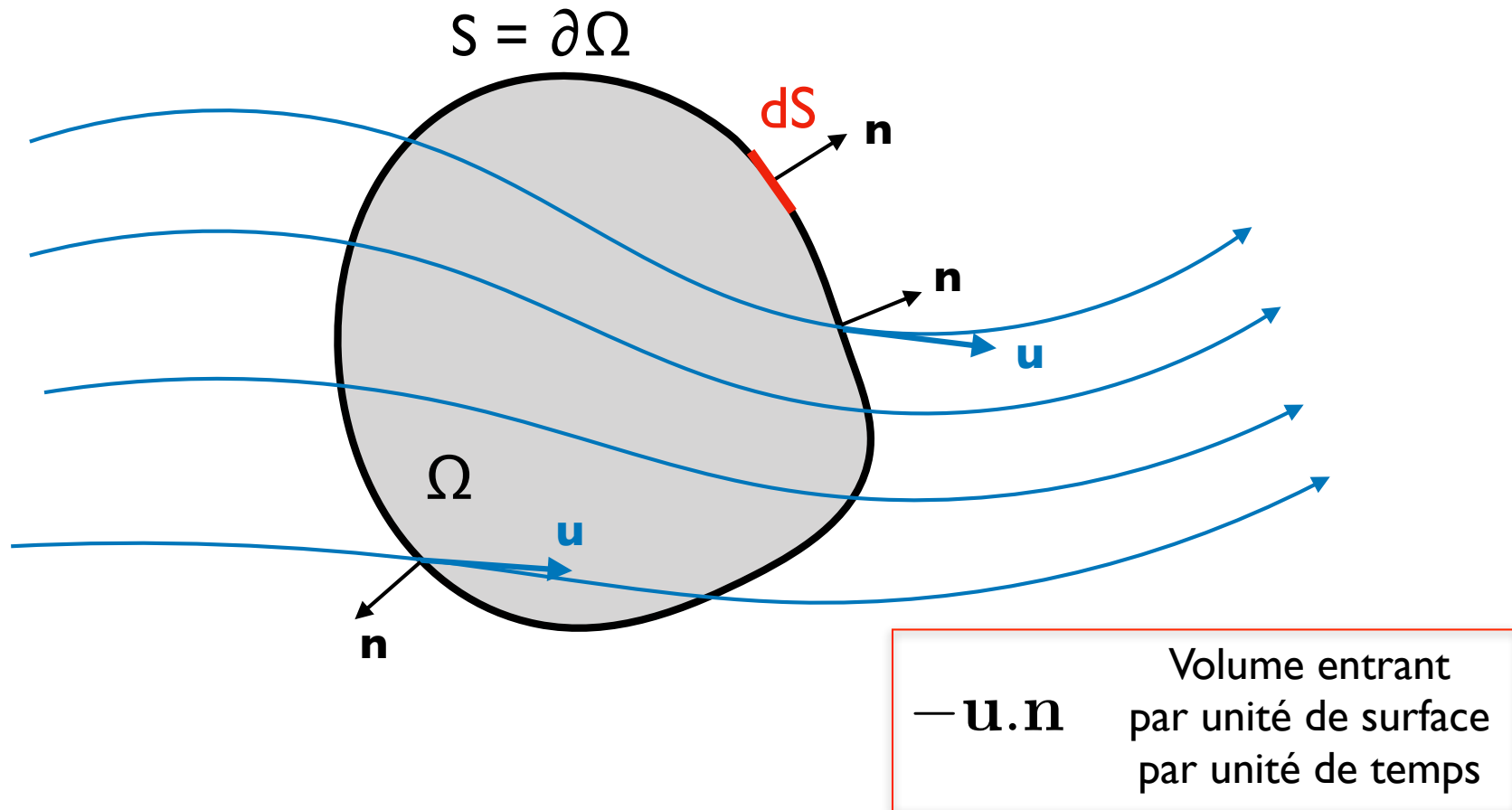
Conservation de la masse dans un écoulement



Conservation de la masse dans un écoulement

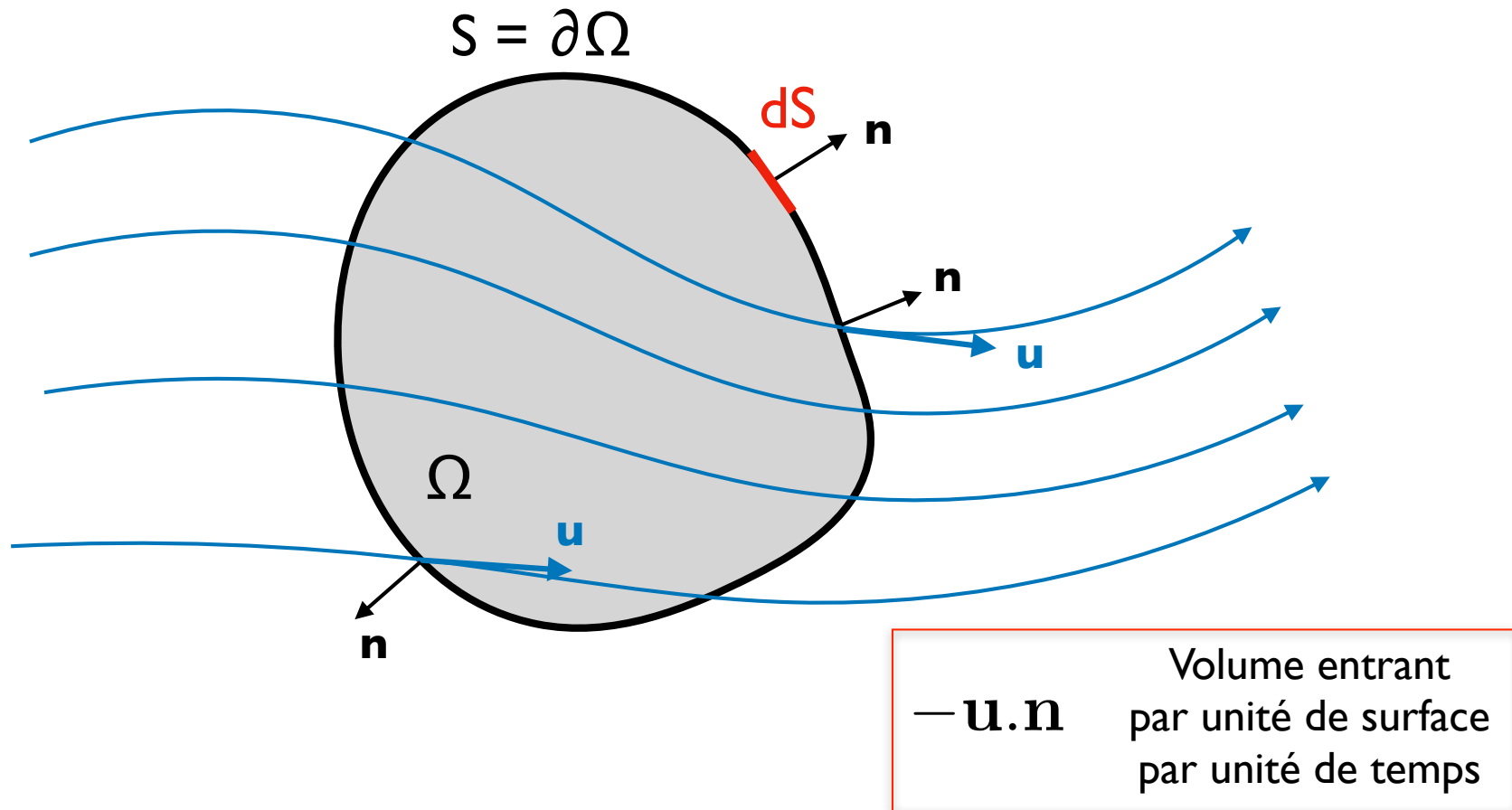


Conservation de la masse dans un écoulement



$$-\int_{\partial\Omega} \rho \mathbf{u} \cdot \mathbf{n} dS + Q$$

Conservation de la masse dans un écoulement



$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV = - \int_{\partial\Omega} \rho \mathbf{u} \cdot \mathbf{n} dS + Q$$

$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV = - \int_{\partial \Omega} \rho \mathbf{u} \cdot \mathbf{n} \, dS + Q$$

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$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV + \int_{\Omega} \nabla \cdot (\rho \mathbf{u}) dV = Q$$

$\nabla \cdot \equiv$ divergence


$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV = - \int_{\partial \Omega} \rho \mathbf{u} \cdot \mathbf{n} dS + Q$$

$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV + \int_{\Omega} \nabla \cdot (\rho \mathbf{u}) dV = Q$$

En l'absence de source de masse

$\nabla \cdot \equiv$ divergence


$$\frac{\partial \rho}{\partial t} + \rho \nabla \cdot \mathbf{u} + \mathbf{u} \cdot \nabla \rho = 0$$

$\nabla \equiv$ gradient

$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV = - \int_{\partial \Omega} \rho \mathbf{u} \cdot \mathbf{n} dS + Q$$

$$\int_{\Omega} \frac{\partial \rho}{\partial t} dV + \int_{\Omega} \nabla \cdot (\rho \mathbf{u}) dV = Q$$

En l'absence de source de masse

$\nabla \cdot \equiv$ divergence


$$\frac{\partial \rho}{\partial t} + \rho \nabla \cdot \mathbf{u} + \mathbf{u} \cdot \nabla \rho = 0$$

$\nabla \equiv$ gradient

Si le fluide est « incompressible », masse volumique constante

$$\nabla \cdot \mathbf{u} = 0$$

Fluide comme “incompressible” ?

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$$\frac{\delta \rho}{\rho} = \frac{1}{\rho} \frac{\partial \rho}{\partial p} \delta p$$

Fluide comme “incompressible” ?

$$\frac{\delta \rho}{\rho} = \frac{1}{\rho} \frac{\partial \rho}{\partial p} \delta p$$

$$\frac{\delta \rho}{\rho} = \frac{1}{\rho c^2} \delta p$$

c : vitesse du son

Fluide comme “incompressible” ?

$$\frac{\delta \rho}{\rho} = \frac{1}{\rho} \frac{\partial \rho}{\partial p} \delta p$$

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c : vitesse du son

En écoulement dominé par l’inertie du fluide: $\delta p \sim \rho u^2$

$$\frac{\delta \rho}{\rho} \sim \frac{u^2}{c^2} = M^2$$

M : Nombre de Mach

Fluide comme “incompressible” ?

$$\frac{\delta \rho}{\rho} = \frac{1}{\rho} \frac{\partial \rho}{\partial p} \delta p$$

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En écoulement dominé par l'inertie du fluide: $\delta p \sim \rho u^2$

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M : Nombre de Mach

$$M \ll 1$$

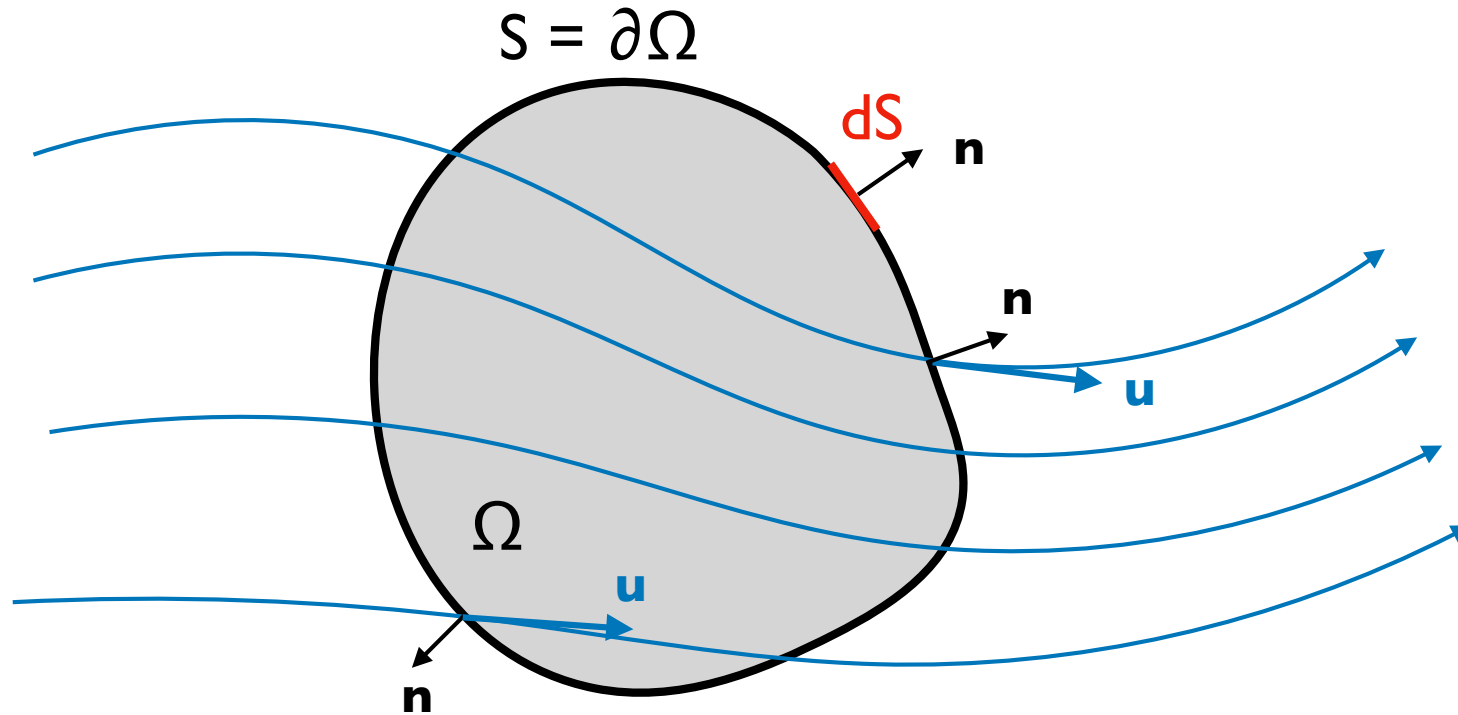
Fluide quasi incompressible



$M \sim 1$ ou $M > 1$

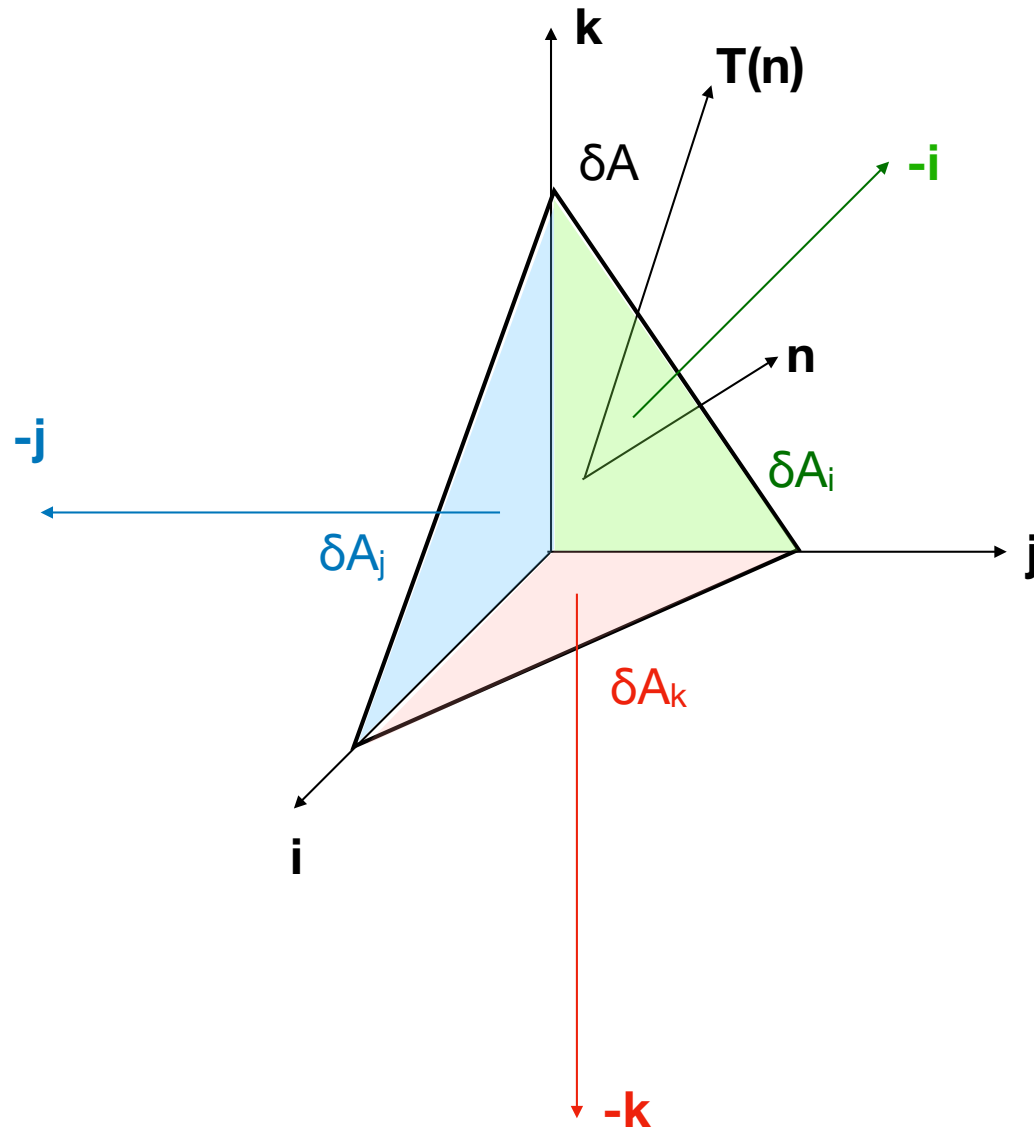
Ondes de choc

Conservation de la quantité de mouvement



$$\int_{\Omega} \frac{\partial \rho \mathbf{u}}{\partial t} dV = \int_{\Omega} \rho \mathbf{f} dV - \int_{\partial\Omega} \underbrace{\rho \mathbf{u}}_{\text{vecteur}} \underbrace{\mathbf{u} \cdot \mathbf{n}}_{\text{scalaire}} dS + \int_{\partial\Omega} \underbrace{\underline{\underline{\sigma}} \cdot \mathbf{n}}_{\text{contraintes}} dS$$

T(n) : vecteur contrainte totale sur la face de normale **n**



Forces de surface

$$\mathbf{T}(\mathbf{n})\delta A$$

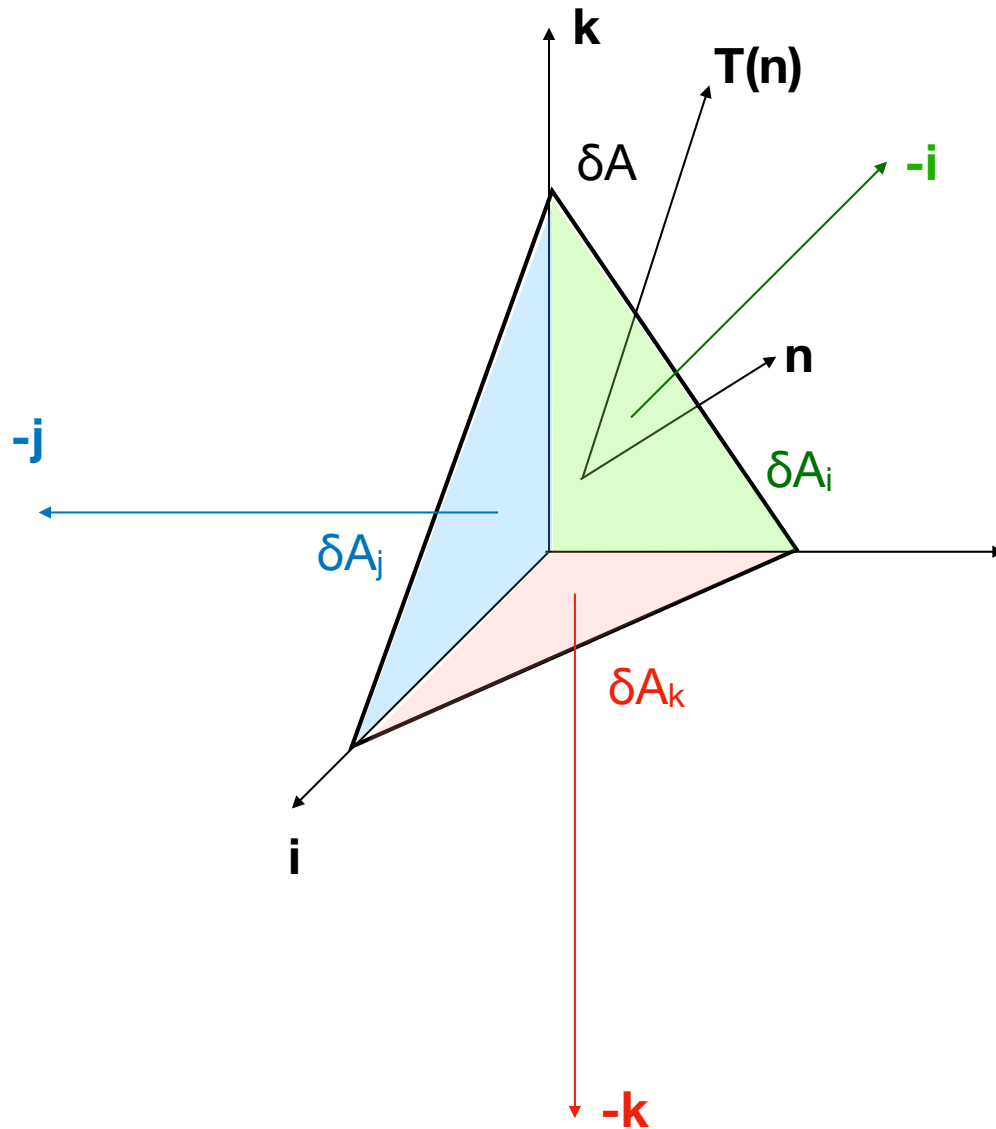
$$\mathbf{T}(-\mathbf{i})\delta A_i$$

$$\mathbf{T}(-\mathbf{j})\delta A_j$$

$$\mathbf{T}(-\mathbf{k})\delta A_k$$

$$\mathbf{T}(-\mathbf{i}) = -\mathbf{T}(\mathbf{i})$$

$$\delta A_i = \delta A \mathbf{i} \cdot \mathbf{n}$$



Forces de surface

$$\mathbf{T}(\mathbf{n}) \delta A$$

$$-\mathbf{T}(\mathbf{i}) \mathbf{i} \cdot \mathbf{n} \delta A$$

$$-\mathbf{T}(\mathbf{j}) \mathbf{j} \cdot \mathbf{n} \delta A$$

$$-\mathbf{T}(\mathbf{k}) \mathbf{k} \cdot \mathbf{n} \delta A$$

$$\mathbf{F}_{\text{surface}} = \sum \mathbf{T} \delta A$$

pfd sur l'élément de volume :

$$\rho \delta V \mathbf{a} = \rho \mathbf{f} \delta V + \sum \mathbf{T} \delta A$$

$$\rho \mathbf{a} = \rho \mathbf{f} + \sum \mathbf{T} \frac{\delta A}{\delta V}$$

$$\frac{\delta A}{\delta V} \sim \frac{1}{L} \rightarrow \infty \quad L \rightarrow 0$$

$$\sum \mathbf{T} = 0$$

$$\mathbf{T}(\mathbf{n}) = \mathbf{T}(\mathbf{i}) \mathbf{i} \cdot \mathbf{n} + \mathbf{T}(\mathbf{j}) \mathbf{j} \cdot \mathbf{n} + \mathbf{T}(\mathbf{k}) \mathbf{k} \cdot \mathbf{n}$$

$$\mathbf{T}(\mathbf{n}) = \mathbf{T}(\mathbf{i}) n_x + \mathbf{T}(\mathbf{j}) n_y + \mathbf{T}(\mathbf{k}) n_z$$

Composante x $T_x(\mathbf{n}) = T_x(\mathbf{i}) n_x + T_x(\mathbf{j}) n_y + T_x(\mathbf{k}) n_z$

$$T_x(\mathbf{n}) = \sigma_{xx} n_x + \sigma_{xy} n_y + \sigma_{xz} n_z$$

$$\sigma = \begin{pmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{pmatrix}$$

contraintes normales

contraintes tangentielles

Tenseur des contraintes

$$\begin{pmatrix} T_x \\ T_y \\ T_z \end{pmatrix} = \begin{pmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{pmatrix} \begin{pmatrix} n_x \\ n_y \\ n_z \end{pmatrix}$$

Vecteur contrainte
(force/unité de
surface)

Tenseur des
contraintes

Normale à la
surface considérée

$$\mathbf{T} = \sigma \cdot \mathbf{n}$$

Conservation de la quantité de mouvement

$$\int_{\Omega} \frac{\partial \rho \mathbf{u}}{\partial t} dV = - \int_{\partial \Omega} \rho \mathbf{u} \mathbf{u} \cdot \mathbf{n} dS + \int_{\Omega} \rho \mathbf{f} dV + \int_{\partial \Omega} \boldsymbol{\sigma} \cdot \mathbf{n} dS$$

Théorème de la divergence:

$$\int_{\Omega} \frac{\partial \rho \mathbf{u}}{\partial t} dV + \int_{\Omega} \nabla \cdot \rho \mathbf{u} \mathbf{u} dV = \int_{\Omega} \rho \mathbf{f} dV + \int_{\Omega} \nabla \cdot \boldsymbol{\sigma} dV$$

$\rho \mathbf{u} \mathbf{u}$: Flux de quantité de mouvement,

↑
produit tensoriel $\longrightarrow$ tenseur de composantes $\rho u_i u_j$

Sa divergence :
(somme sur j)

$$\frac{\partial}{\partial x_j} (\rho u_i u_j) = u_i \frac{\partial \rho u_j}{\partial x_j} + \rho u_j \frac{\partial u_i}{\partial x_j}$$

Divergence des contraintes

$$\nabla . \sigma = \sum_j \frac{\partial \sigma_{ij}}{\partial x_j}$$

Conservation de la quantité de mouvement

Pour un volume élémentaire de fluide :

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \frac{\partial \rho}{\partial t} + \mathbf{u} \nabla \cdot \rho \mathbf{u} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = \rho \mathbf{f} + \nabla \cdot \sigma$$

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \underbrace{\mathbf{u} \left(\frac{\partial \rho}{\partial t} + \nabla \cdot \rho \mathbf{u} \right)}_{= 0} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = \rho \mathbf{f} + \nabla \cdot \sigma$$

Conservation de la masse

$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = \rho \mathbf{f} + \nabla \cdot \sigma$$

$$m \mathbf{a} = \mathbf{F}$$

Accélération d'un élément de fluide

$$m\mathbf{a} = \mathbf{F}$$

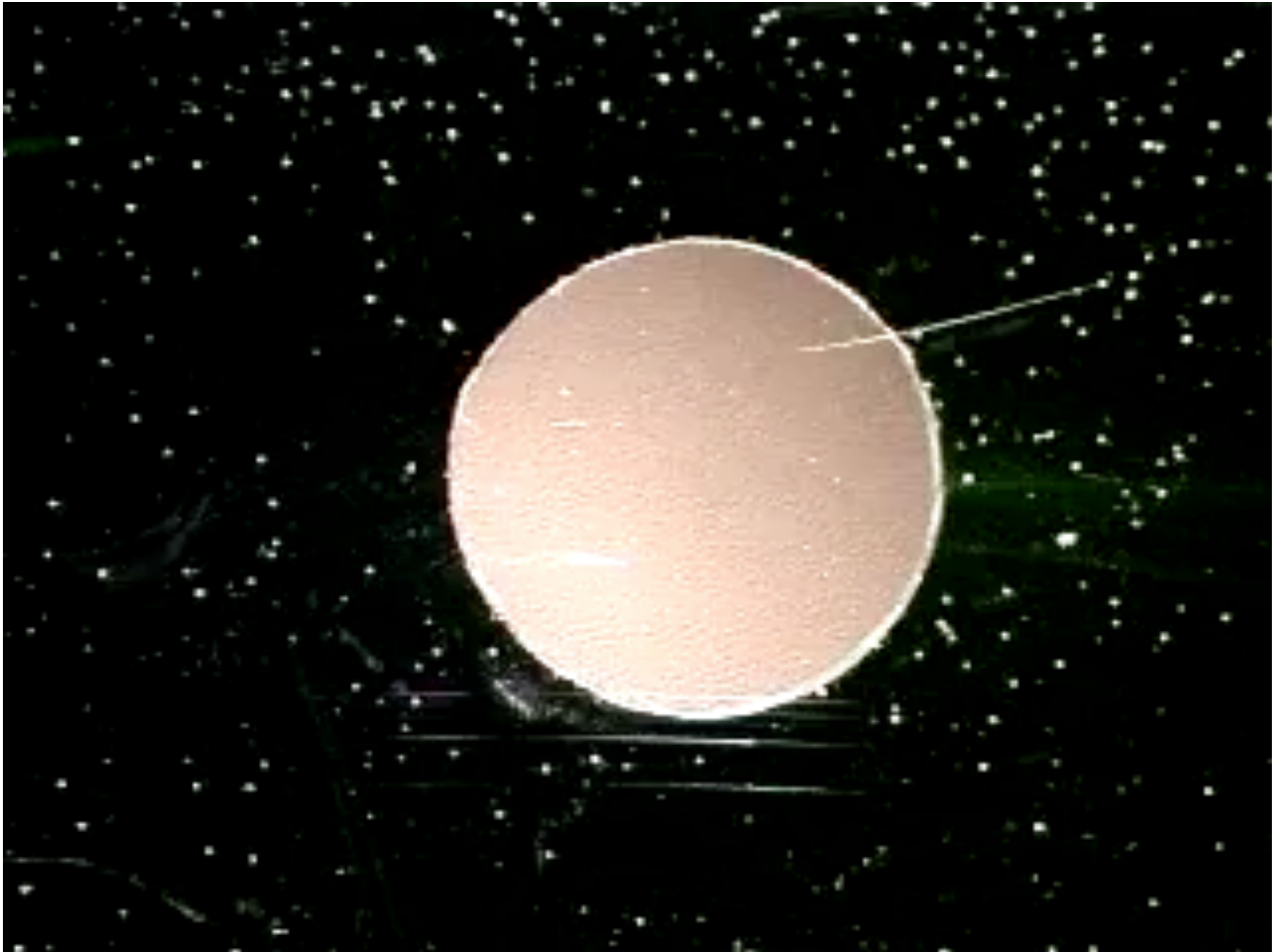
$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = \rho \mathbf{f} + \nabla \cdot \sigma$$

$$\mathbf{a} = \boxed{\frac{\partial \mathbf{u}}{\partial t}} + \boxed{\mathbf{u} \cdot \nabla \mathbf{u}}$$

instationnarité

accélération convective

un écoulement stationnaire où $\mathbf{u} \cdot \nabla \mathbf{u} \neq 0$



Écriture de l'accélération convective

Pour la composante i :

$$u_j \frac{\partial u_i}{\partial x_j} = \sum_{j=1,3} u_j \frac{\partial u_i}{\partial x_j}$$

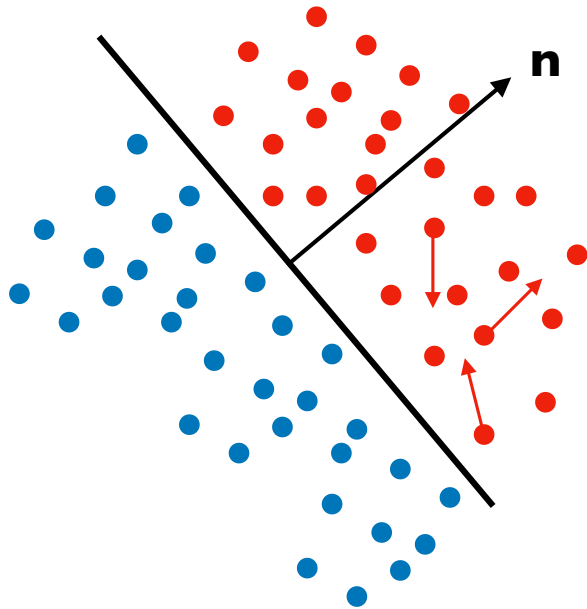
Pour la composante x :

$$u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z}$$

Variation spatiale de u_x projetée sur la vitesse (u_x, u_y, u_z)

Les contraintes dans un fluide

Dans un fluide à l'équilibre, sans écoulement macroscopique



La résultante des interactions entre atomes ou molécules est une **pression isotrope**

$$\sigma = \begin{pmatrix} -p & 0 & 0 \\ 0 & -p & 0 \\ 0 & 0 & -p \end{pmatrix}$$

$$\sigma_{ij} = -p \delta_{ij}$$

$$\nabla \cdot \sigma = -\nabla p = \begin{pmatrix} -\frac{\partial p}{\partial x} \\ -\frac{\partial p}{\partial y} \\ -\frac{\partial p}{\partial z} \end{pmatrix}$$

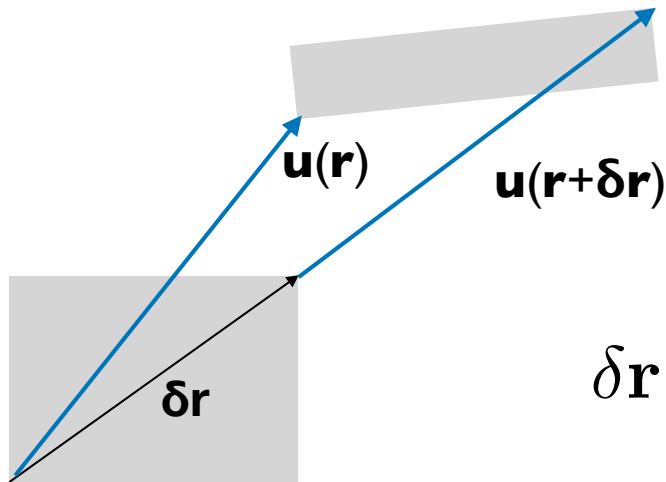
Dans un fluide à l'équilibre, sans écoulement macroscopique

$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = \rho \mathbf{f} + \nabla \cdot \sigma$$

$$0 = \rho \mathbf{g} - \nabla p$$

$$p = p_0 - \rho g z$$

Les déformations d'un élément dans un fluide en écoulement



$$\delta \mathbf{r} = \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix}$$

$$u_x(\mathbf{r} + \delta \mathbf{r}) = u_x(\mathbf{r}) + \frac{\partial u_x}{\partial x} \delta r_x + \frac{\partial u_x}{\partial y} \delta r_y$$

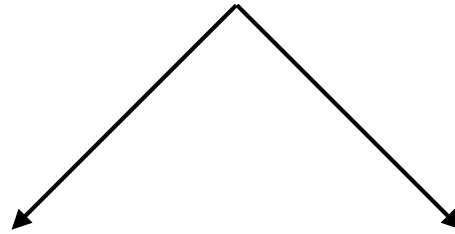
$$u_y(\mathbf{r} + \delta \mathbf{r}) = u_y(\mathbf{r}) + \frac{\partial u_y}{\partial x} \delta r_x + \frac{\partial u_y}{\partial y} \delta r_y$$

$$\begin{pmatrix} \delta u_x \\ \delta u_y \end{pmatrix} = \begin{pmatrix} \frac{\partial u_x}{\partial x} & \frac{\partial u_x}{\partial y} \\ \frac{\partial u_y}{\partial x} & \frac{\partial u_y}{\partial y} \end{pmatrix} \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix}$$

$$\delta \mathbf{u} = \nabla \mathbf{u} \cdot \delta \mathbf{r}$$

Décomposition du gradient de vitesse

$$\begin{pmatrix} \delta u_x \\ \delta u_y \end{pmatrix} = \begin{pmatrix} \frac{\partial u_x}{\partial x} & \frac{\partial u_x}{\partial y} \\ \frac{\partial u_y}{\partial x} & \frac{\partial u_y}{\partial y} \end{pmatrix} \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix}$$



$$\begin{bmatrix} \frac{\partial u_x}{\partial x} & \frac{1}{2} \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right) \\ \frac{1}{2} \left(\frac{\partial u_y}{\partial x} + \frac{\partial u_x}{\partial y} \right) & \frac{\partial u_y}{\partial y} \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 & \frac{\partial u_x}{\partial y} - \frac{\partial u_y}{\partial x} \\ \frac{\partial u_y}{\partial x} - \frac{\partial u_x}{\partial y} & 0 \end{bmatrix}$$

Partie symétrique du
gradient de vitesse

Partie antisymétrique
du gradient de vitesse

Décomposition de la partie symétrique

$$\begin{bmatrix} \frac{\partial u_x}{\partial x} & \frac{1}{2} \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right) \\ \frac{1}{2} \left(\frac{\partial u_y}{\partial x} + \frac{\partial u_x}{\partial y} \right) & \frac{\partial u_y}{\partial y} \end{bmatrix}$$

$$\frac{1}{2} \begin{bmatrix} \frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} & 0 \\ 0 & \frac{\partial u_y}{\partial y} + \frac{\partial u_x}{\partial x} \end{bmatrix} = \frac{1}{2} \nabla \cdot \mathbf{u} \begin{pmatrix} 1 & 0 \\ 0 & 1 \end{pmatrix}$$

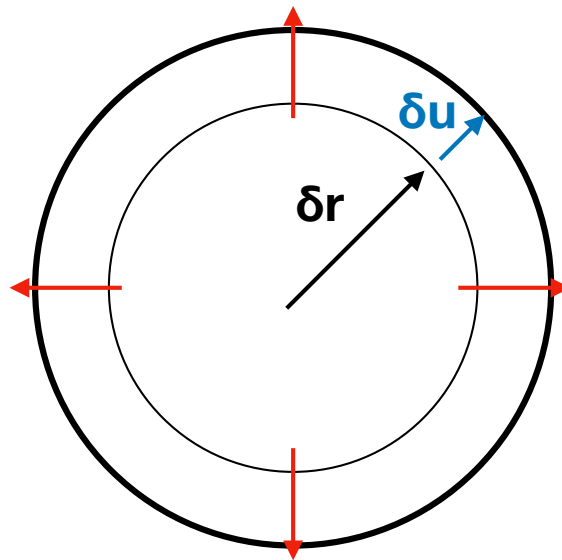
Sphérique

$$\frac{1}{2} \begin{bmatrix} \frac{\partial u_x}{\partial x} - \frac{\partial u_y}{\partial y} & \frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \\ \frac{\partial u_y}{\partial x} + \frac{\partial u_x}{\partial y} & \frac{\partial u_y}{\partial y} - \frac{\partial u_x}{\partial x} \end{bmatrix} = \begin{pmatrix} a & b \\ b & -a \end{pmatrix}$$

Déviateur (trace nulle)

Effet de la partie sphérique

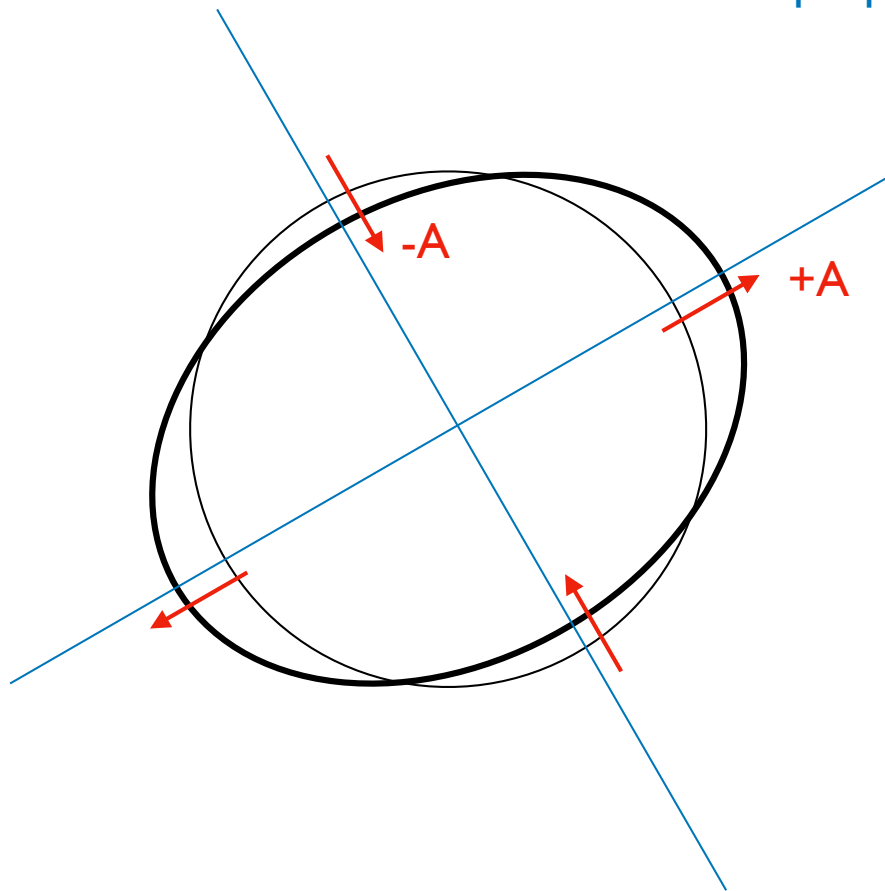
$$\begin{pmatrix} \delta u_x \\ \delta u_y \end{pmatrix} = \frac{1}{2} \nabla \cdot \mathbf{u} \begin{pmatrix} 1 & 0 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix} = \frac{1}{2} \nabla \cdot \mathbf{u} \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix}$$



Effet du déviateur

$$\begin{pmatrix} \delta u_x \\ \delta u_y \end{pmatrix} = A \begin{pmatrix} 1 & 0 \\ 0 & -1 \end{pmatrix} \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix} = A \begin{pmatrix} \delta r_x \\ -\delta r_y \end{pmatrix}$$

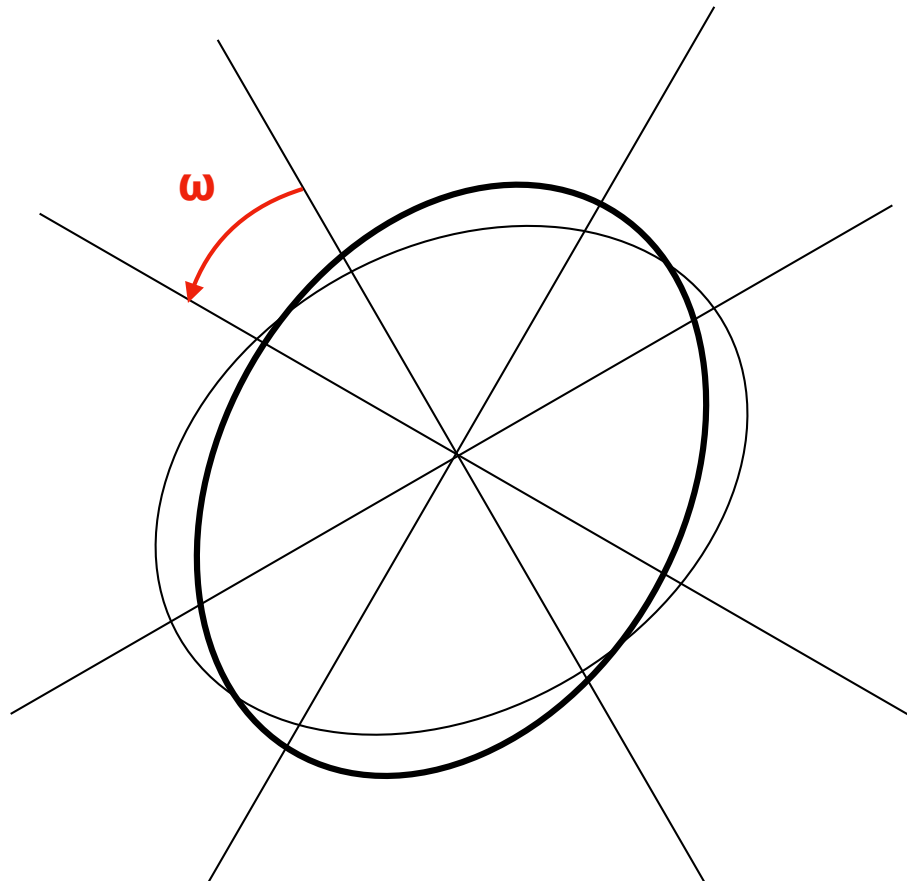
Dans les axes propres



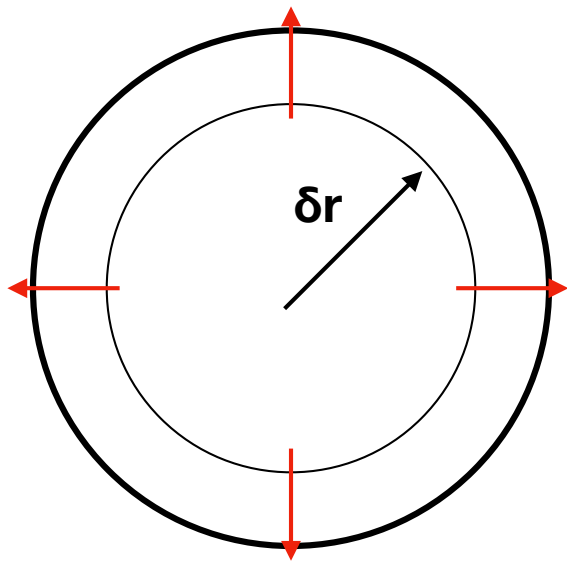
Effet de la partie antisymétrique

$$\begin{pmatrix} \delta u_x \\ \delta u_y \end{pmatrix} = \omega \begin{pmatrix} 0 & -1 \\ 1 & 0 \end{pmatrix} \begin{pmatrix} \delta r_x \\ \delta r_y \end{pmatrix} = \omega \begin{pmatrix} -\delta r_y \\ \delta r_x \end{pmatrix}$$

$$\delta \mathbf{u} \perp \delta \mathbf{r}$$



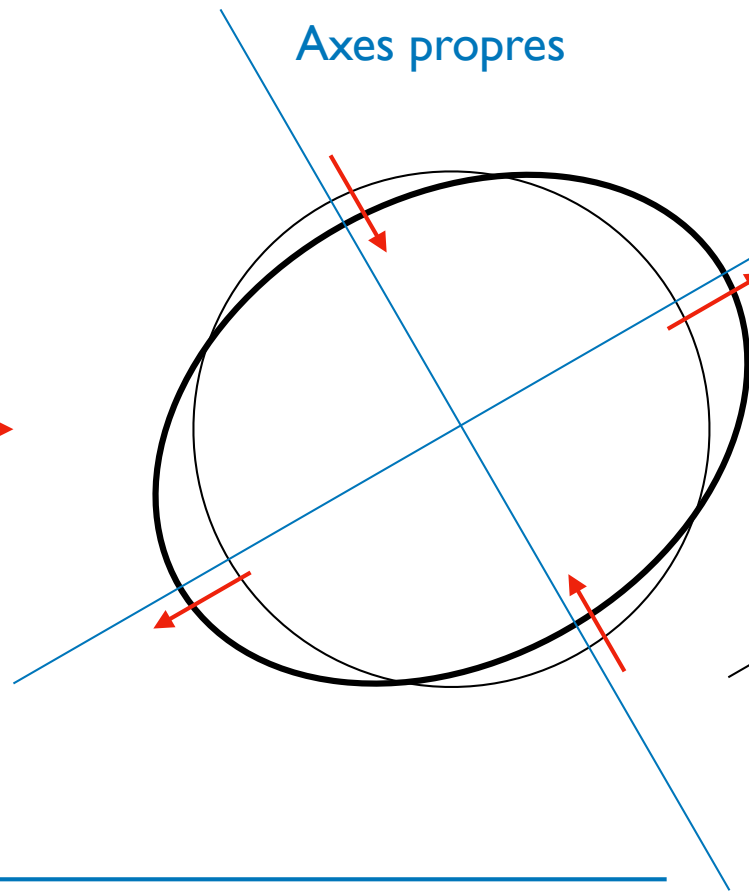
Axes propres



Partie symétrique du
gradient de vitesse

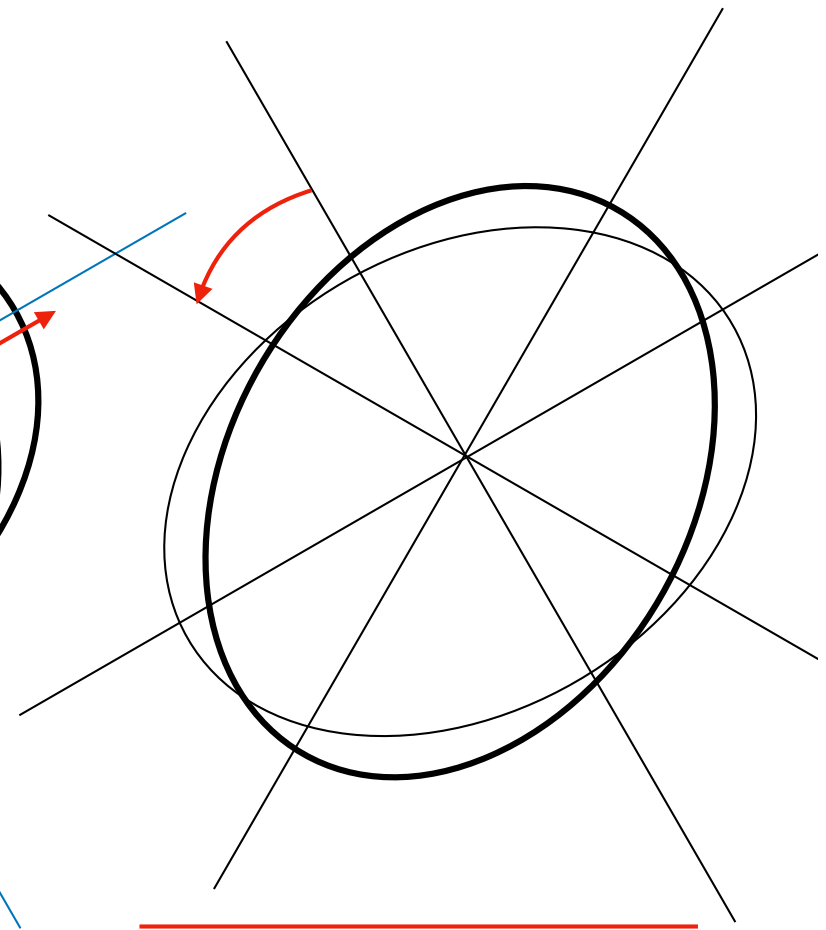
Sphérique

Chgt. de volume
si $\text{div } \mathbf{u} \neq 0$



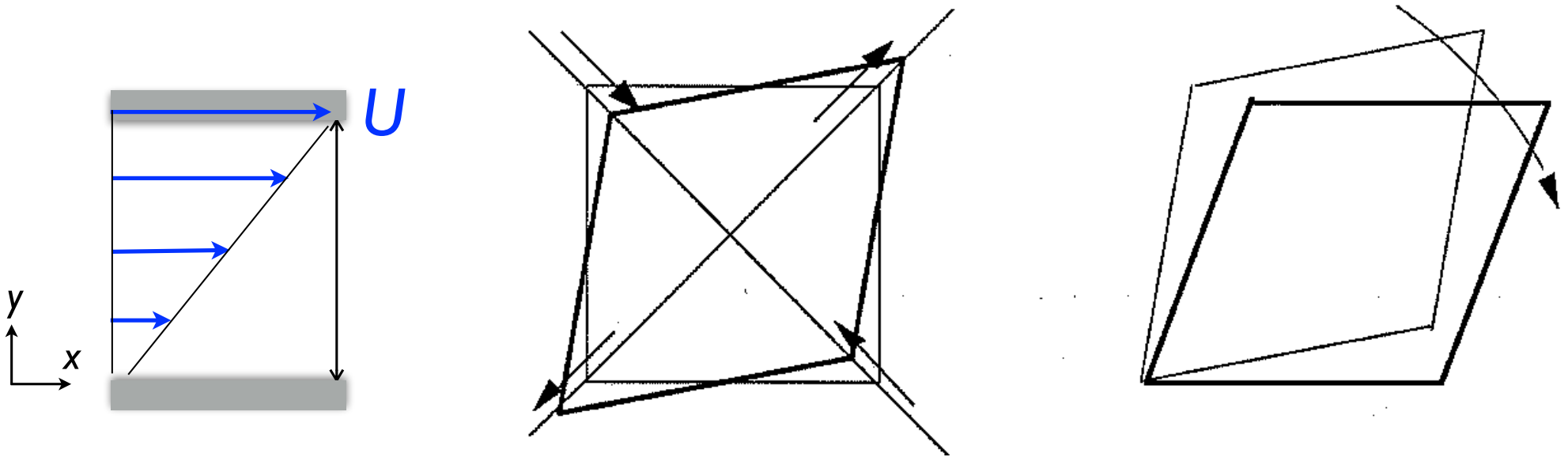
Déviateur

Déformation à
volume constant



Partie antisymétrique
du gradient de vitesse
Rotationnel

Rotation sans
déformation



Ecoulement de cisaillement simple $u_x = Gy$
 Décomposition en déformation pure suivant des axes propres et rotation

$$\nabla \mathbf{u} = \begin{pmatrix} 0 & G \\ 0 & 0 \end{pmatrix} = \frac{1}{2} \begin{pmatrix} 0 & G \\ G & 0 \end{pmatrix} + \frac{1}{2} \begin{pmatrix} 0 & G \\ -G & 0 \end{pmatrix}$$

Les contraintes dans un fluide newtonien

Comportement mécanique du fluide caractérisé par une seule quantité :

sa viscosité dynamique η

Contraintes : fonctions linéaires de la partie symétrique du gradient de vitesse

partie symétrique du gradient de vitesse : déformation d'un élément de fluide

partie antisymétrique du gradient de vitesse : rotation en bloc d'un élément de fluide

Les contraintes dans un fluide newtonien

$$\sigma_{ij} = -p \delta_{ij} + 2 \eta e_{ij}$$

$$\underline{\underline{e}} = \frac{1}{2}(\nabla \mathbf{u} + \nabla \mathbf{u}^T)$$

$$\sigma = \begin{pmatrix} -p + 2\eta \frac{\partial u_x}{\partial x} & \eta \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right) & \eta \left(\frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right) \\ \eta \left(\frac{\partial u_x}{\partial y} + \frac{\partial u_y}{\partial x} \right) & -p + 2\eta \frac{\partial u_y}{\partial y} & \eta \left(\frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right) \\ \eta \left(\frac{\partial u_x}{\partial z} + \frac{\partial u_z}{\partial x} \right) & \eta \left(\frac{\partial u_y}{\partial z} + \frac{\partial u_z}{\partial y} \right) & -p + 2\eta \frac{\partial u_z}{\partial z} \end{pmatrix}$$

$$\nabla \cdot \sigma = -\nabla p + \eta \Delta \mathbf{u}$$

Dans un fluide **newtonien**, avec **écoulement macroscopique**

$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + (\mathbf{u} \cdot \nabla) \mathbf{u} \right) = \rho \mathbf{f} + \nabla \cdot \sigma$$

$$\nabla \cdot \sigma = -\nabla p + \eta \Delta \mathbf{u}$$

Équation de Navier-Stokes

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} = -\nabla p + \eta \Delta \mathbf{u} + \rho \mathbf{f}$$

$$\frac{\partial \mathbf{u}}{\partial t} + (\mathbf{u} \cdot \nabla) \mathbf{u} = -\frac{1}{\rho} \nabla p + \nu \Delta \mathbf{u} + \mathbf{f}$$

viscosité cinématique $\nu = \eta / \rho$

L'équation de Navier-Stokes dans toute sa splendeur (horreur ?)

$$\frac{\partial u_x}{\partial t} + u_x \frac{\partial u_x}{\partial x} + u_y \frac{\partial u_x}{\partial y} + u_z \frac{\partial u_x}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u_x}{\partial x^2} + \frac{\partial^2 u_x}{\partial y^2} + \frac{\partial^2 u_x}{\partial z^2} \right) + f_x$$

$$\frac{\partial u_y}{\partial t} + u_x \frac{\partial u_y}{\partial x} + u_y \frac{\partial u_y}{\partial y} + u_z \frac{\partial u_y}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left(\frac{\partial^2 u_y}{\partial x^2} + \frac{\partial^2 u_y}{\partial y^2} + \frac{\partial^2 u_y}{\partial z^2} \right) + f_y$$

$$\frac{\partial u_z}{\partial t} + u_x \frac{\partial u_z}{\partial x} + u_y \frac{\partial u_z}{\partial y} + u_z \frac{\partial u_z}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left(\frac{\partial^2 u_z}{\partial x^2} + \frac{\partial^2 u_z}{\partial y^2} + \frac{\partial^2 u_z}{\partial z^2} \right) + f_z$$

Comment la simplifier ?

Une analyse dimensionnelle de l'équation de Navier-Stokes

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla) \mathbf{u} = -\nabla p + \eta \Delta \mathbf{u} + \rho \mathbf{f}$$

$$\mathbf{u} \sim U$$

$$\nabla \sim 1/L$$

$$\Delta \sim 1/L^2$$

Échelle de longueur
pertinente qui caractérise la
variation spatiale de vitesse

$$\rho \mathbf{u} \cdot \nabla \mathbf{u} \sim \rho \frac{U^2}{L}$$

Terme inertiel

$$\eta \Delta \mathbf{u} \sim \eta \frac{U}{L^2}$$

Terme visqueux

$$\frac{\rho \mathbf{u} \cdot \nabla \mathbf{u}}{\eta \Delta \mathbf{u}} \sim \frac{\rho U L}{\eta} = \frac{U L}{\nu}$$

Nombre de Reynolds

